

Если выдѣлить двумя безконечно близкими поперечными сѣченіями элементъ стержня длиной δx , то силы инерціи, соответствующія вращенію этого элемента, дадутъ намъ моменты равныя

$$-\frac{J\gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2} \delta x \dots \dots \dots (c)$$

При составленіи дифференціального ур-ія для изогнутой оси стержня нужно принять во вниманіе не только непрерывно распределенную нагрузку, соответствующую силамъ инерціи поступательнаго движенія, но и непрерывно распределенные моменты, опредѣляемые ϕ -ой (c). Въ такомъ случаѣ извѣстнае изгибающее моментъ вдоль оси стержня представляется такъ

$$\frac{\partial M}{\partial x} = Q - \frac{J\gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2}$$

Здѣсь Q поперечная сила, получающаяся отъ силъ инерціи поступательнаго движенія элементовъ стержня.

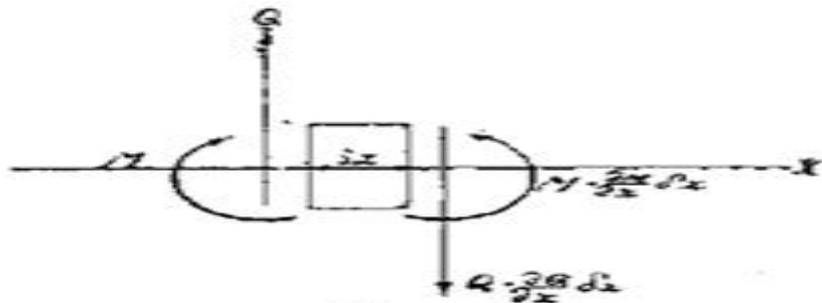
Вставляя найденное значеніе производной $\frac{\partial M}{\partial x}$ въ дифференціальное ур-іе

$$EJ \frac{\partial^2 y}{\partial x^2} = - \frac{\partial^2 M}{\partial x^2},$$

получаемъ

$$EJ \frac{\partial^2 y}{\partial x^2} = - \frac{\gamma F^2}{g} \frac{\partial^2 y}{\partial s^2} + \frac{J\gamma}{g} \frac{\partial^4 y}{\partial x^2 \partial t^2} \dots \dots \dots (177)$$

Второй членъ въ правой части ур-ія представляетъ собой поправку, обусловленную поворотомъ поперечныхъ сѣченій при колебаніяхъ.



Черт. 76.

Волѣе точное ур-іе мы получимъ, если примемъ во вниманіе не только вліяніе силъ инерціи, соответствующихъ повороту сѣченій, но также и вліяніе касательныхъ напряженій на прогибъ стержня¹⁾. Выдѣлимъ изъ стержня двумя безконечно близкими сѣченіями элементъ длиной δx и составимъ для него ур-іе движенія. Положеніе выдѣленнаго элемента опредѣлится перемѣщеніемъ y его центра тяжести и угломъ поворота ϕ относительно нейтральной линіи²⁾. Такъ какъ прогибъ

стержня обусловленъ не только относительнымъ поворотомъ поперечныхъ сѣченій но также и движеньемъ ихъ одно относительно другого, то уголъ наклона касе-

¹⁾ Вліяніе касательныхъ напряженій на періодъ поперечныхъ колебаній стержня обыкновенно не принимается въ расчетъ, а между тѣмъ соответствующая ему поправка по своей величинѣ больше той, которая получается отъ поперечнаго сѣченія.

²⁾ При составленіи ур-ія мы рассматриваемъ элементъ, какъ абсолютно твердое тѣло и пренебрегаемъ искривленіемъ поперечныхъ сѣченій, поэтому наша поправка лишь приближенно оцѣниваетъ вліяніе касательныхъ силъ. Намъ совѣстно съ проф. П. Эрихнстомъ получить также точное рѣшеніе задачи для колеса прямоугольнаго поперечнаго сѣченія. Это рѣшеніе, какъ и цитированное нами выше (стр. 184) рѣшеніе Ресншаммера для круглаго цилиндра, показываютъ, что приведенное здѣсь приближенное рѣшеніе съ достаточной точностью учитываетъ вліяніе касательныхъ силъ.

Elasticity Timoshenko Solution Manual

Shasha Hu



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