

Если выдвинуть двумя бесконечно близкими поперечными сечениями элемент стержня длиной  $\delta x$ , то силы инерции, соответствующие вращению этого элемента, дадут нам моменты равный

$$-\frac{J\gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2} \delta x \dots \dots \dots (c)$$

При составлении дифференциального ур-ия для изогнутой оси стержня нужно принять во внимание не только непрерывно распределенную нагрузку, соответствующую силам инерции поступательного движения, но и непрерывно распределенные моменты, определяемые  $\phi$ -ой (c). В таком случае влияние изгибающего момента вдоль оси стержня представляется такъ

$$\frac{\partial M}{\partial x} = Q - \frac{J\gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2}$$

Здѣсь  $Q$  поперечная сила, получающаяся отъ силъ инерции поступательного движения элементовъ стержня.

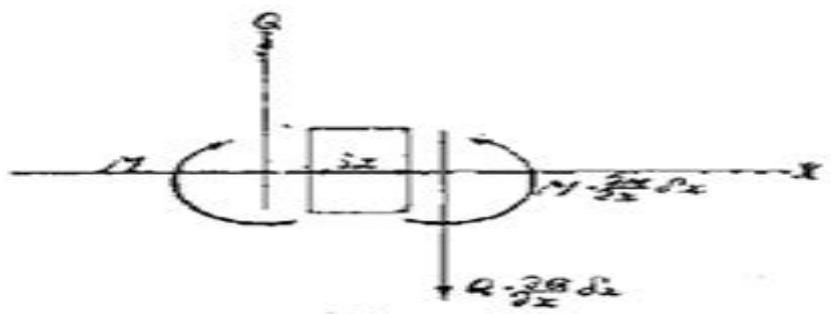
Вставляя найденное значение производной  $\frac{\partial M}{\partial x}$  въ дифференциальное ур-ие

$$EJ \frac{\partial^2 y}{\partial x^2} = - \frac{\partial^2 M}{\partial x^2},$$

получаемъ

$$EJ \frac{\partial^4 y}{\partial x^4} = - \frac{\gamma F^2}{g} \frac{\partial^2 y}{\partial s^2} + \frac{J\gamma}{g} \frac{\partial^4 y}{\partial x^2 \partial t^2} \dots \dots \dots (177)$$

Второй членъ въ правой части ур-ия представляетъ собой поправку, обусловленную поворотомъ поперечныхъ сечений при колебаніяхъ.



Черт. 76.

Волѣе точное ур-іе мы получимъ, если примемъ во вниманіе не только влияние силъ инерции, соответствующихъ повороту сѣченій, но также и влияние касательныхъ напряженій на прогибъ стержня<sup>1)</sup>. Выдѣлимъ изъ стержня двумя бесконечно близкими сѣченіями элементъ длиной  $\delta x$  и составимъ для него ур-іе движенія. Положеніе выдѣленного элемента опредѣлится перемѣщеніемъ  $y$  его центра тяжести и угломъ поворота  $\phi$  относительно нейтральной линіи<sup>2)</sup>. Такъ какъ прогибъ

стержня обусловленъ не только относительнымъ поворотомъ поперечныхъ сѣченій но также и движеньемъ ихъ одно относительно другого, то уголъ наклона касе-

<sup>1)</sup> Вліяніе касательныхъ напряженій на періодъ поперечныхъ колебаній стержня обыкновенно не принимается въ расчетъ, а между тѣмъ соответствующая ему поправка по своей величинѣ больше той, которая получается отъ поперечныхъ сѣченій.

<sup>2)</sup> При составленіи ур-іа мы рассматриваемъ элементъ, какъ абсолютно твердое тѣло и пренебрегаемъ искривленіемъ поперечныхъ сѣченій, поэтому наша поправка лишь приближенно оцѣниваетъ вліяніе касательныхъ силъ. Намъ совѣстно съ проф. П. Эрихтстомъ получить также точное рѣшеніе задачи для колеса прямоугольнаго поперечнаго сѣченія. Это рѣшеніе, какъ и цитированное нами выше (стр. 184) рѣшеніе Реснхаммера для круглаго цилиндра, показываютъ, что приведенное здѣсь приближенное рѣшеніе съ достаточной точностью учитываетъ вліяніе касательныхъ силъ.

# Elasticity Timoshenko Solution Manual

**K Morrison**



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